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# Contents

1	Purpose of this Document	6
2	References	7
3	Description of Terms and Concepts	8
	<ul> <li>Abbreviations</li> <li>Terminology – Torque within the Powertrain Domain .</li> <li>Terminology – Fast and Slow Torque Requests</li> <li>Overview of AUTOSAR torque application interfaces .</li> </ul>	8 8 9 11
4	Architecture Overview	15
5	Description of Exemplary Software Components	17
	<ol> <li>Powertrain Coordinator – PTC (PtCoorr)</li></ol>	17 17 19 19 20 )20 20 20 fty)20 fty)20 cellaneous 20 cellaneous 20
0	Additional Information	22
/ 7 7	<ul> <li>Additional Information</li> <li>Differences between SW-Cs and ECUs</li> <li>Functional safety</li> <li>Powertrain Application Interfaces - Decisions / Assum</li> <li>7.3.1 Scope</li> <li>7.3.2 PTC Composition (PtCoorr)</li> <li>7.3.3 Definition of overboost</li> <li>7.3.4 Coordination at the vehicle level</li> <li>7.3.5 PTC Arbitration between Driver and Chassis torque 42</li> <li>7.3.6 Assumptions on modeling style and naming aspect for powertrain domain</li> </ul>	
4 of 6	Document ID 269: AUTOSAF	R_EXP_AIPowertrain



8 Appendix: Mapping Ports to Display Names - Powertrain Domain .. 48



# **1** Purpose of this Document

This document explains design decisions that lead to the standardized applications interfaces relevant to the Powertrain Domain.

The sensor actuator pattern described in this document is not specific to the powertrain domain but can be applied to other domains too, e.g. the chassis domain.

<u>NOTE:</u> If any information in diagrams or text (or conclusions drawn from them) conflict with the information in [2] or [3] or [3b] and this is not explicitly mentioned the information in [2] or [3] or [3b], resp., should be regarded as definitive.



# 2 References

- [1] SW-C and System Modeling Guide AUTOSAR\_TR\_SW-CModelingGuide
- [2] Table of Application Interfaces AUTOSAR\_MOD\_AITable
- [3] XML Specification of Application Interfaces AUTOSAR\_MOD\_AISpecification
- [3b] Application Interfaces Examples AUTOSAR\_MOD\_AISpecificationExamples
- [4] Explanation of Application Interfaces of the Chassis Domain AUTOSAR\_EXP\_AIChassisExplanation

[5] Unique Names for Documentation, Measurement and Calibration: Modeling and Naming Aspects including Automatic Generation AUTOSAR\_TR\_AIMeasurementCalibrationDiagnostics

- [6] Software Component Template AUTOSAR\_TPS\_SoftwareComponentTemplate
- [7] Standardization Template AUTOSAR\_TPS\_StandardizationTemplate
- [8] ANTLR parser generator V3 http://www.antlr.org
- [9] Virtual Function Bus AUTOSAR\_EXP\_VFB
- [10] Glossary AUTOSAR\_TR\_Glossary



# **3** Description of Terms and Concepts

# 3.1 Abbreviations

For abbreviations used in this document please refer to the keyword list in [2] (as .xls) and in [3] (as .arxml).

Additionally please also refer to [10] for explanation of commonly used terms and abbreviations within AUTOSAR.

# 3.2 Terminology – Torque within the Powertrain Domain



#### Figure 1: Powertrain Torque terminology

Sign definition for torque at clutch / torque at wheels:

Positive value means that torque is transmitted from the engine to the drivetrain / from the powertrain to the wheels.

Negative value means that torque is transmitted from the drivetrain to the engine / from the wheels to the powertrain.



Zero means that no torque is transmitted between engine and drivetrain / between wheels and powertrain.

#### Engine Clutch

For Hybrid Systems an additional clutch can be present between combustion engine and electric machine.

#### 3.3 Terminology – Fast and Slow Torque Requests

Many torque request interfaces have the additional descriptors "Fast" or "Slow".

These descriptors are relevant to gasoline spark ignition engines, whose torque output can be modified by means of throttle angle (and hence air mass) and ignition timing. In general, the torque output responds slowly to changes in throttle angle due to fluid dynamics in the manifold and cyl-inder head. The reaction to ignition timing changes is almost instantaneous, especially at higher engine speeds.

"Fast" refers to the "immediate" / "instant" torque request, typically achieved by ignition timing.

"Slow" refers to the longer term or "torque reserve" request, usually the input to throttle control.

Note that a gasoline engine running at optimum ignition timing cannot **increase** torque quickly as the throttle is the only means for the increase. However, pre-emptively opening the throttle and running with retarded ignition to maintain the the original (lower) torque allows the torque to be increased quickly by ignition a short time in the future. This operation is usually achieved by setting the "Slow" torque request to be greater than the "Fast" torque request to provide this "torque reserve", allowing the torque to be rapidly increased by increasing the "Fast" request.





Figure 2: The Torque Reserve concept, with "Fast" and "Slow" torque requests

For conventional diesel engines only the fast torque interfaces are relevant. However, future diesel engines could have the possibility to use both fast and slow torque interfaces.



# 3.4 Overview of AUTOSAR torque application interfaces

Legend within Figure 3:

<ShortName of Powertrain Port>/ <ShortName of Chassis Port>] <LongName of Port>

Note: Obsolete Interfaces from before AUTOSAR 4.1.1 are still included.

	Overview of AUTOSAR	crankshaft tor	queinterfaces
Crankshaf	ttorque		
	Upper limit of data type     TergTqCrksttMaxOptnCdn     Maximum Engine Torque at Crankshaft at Optimum Conditions     EngTqCrksttMax     Maximum Engine Torque at Crankshaft	Maximum air mass at current speed	Torque intervention EngTqCrksftMaxProtn
	EngTqCrksftMaxFast Maximum Engine Torque at Crankshaft Fast Path	≤Setpoint	
	EngTqCrksft (spark retard, Actual Engine Torque at Crankshaft lambda etc.) Maximu spark ret	m Currentairmass and speed	
	EngTqCrksftMinFast Minimum Engine Torque at Crankshaft Fast Path	▲-)Setpoint	
0 -	EngTqCrksftMinBasc Minimum Engine Torque at Crankshaft for Powertrain realized by Slow Pat	<u>h</u>	)
Indicated torque	EngTqCrksftMinWoCutOff Minimum Engine Torque at Crankshaft for Powertrain realized by Slow and	East Path	Minimum air mass at current speed
l	EngTqCrksftMin Minimum Engine Torque at Crankshaft considering all engine losses		
			Additional interfaces:
			<b>PtTqResvEng</b> Torque Reserve Request from Powertrain to Engine
			<b>EngTqResvPt</b> Torque Reserve Request from Engine to Powertrain
	Lower limit of data type		

Figure 3: Overview of AUTOSAR crankshaft torque interfaces



Upper limit of data type	
PtTqCluMaxDrv Powertrain: Maximum Available Torque at Clutch for Driver	TrsmTqCluMaxProtn Transmission: Maximum Torque at Clutch Requested by Transmission for Gearbox Protection
PtTqClu /EngTqActAtClu       DrvReqTqCluSlow// Driver Request of Cu         Powertrain: Actual Torque at Clutch       Setpoint       DrvReqTqCluFast. T         PtTqCluDyn       EngTqDynJ       Driver Request of cu         Powertrain:       EngTqDynJ       Engine dynamic moment of inertia f	Transmission: Maximum Torque at Clutch Requested by Transmission: Maximum Torque at Clutch Requested by Transmission for Shift Energy Management on Fast Path TqSlowACLUReqdByDrvr tch Torque (SlowTorque Path) TqFastAtCluReqdByDrvr tch Torque (Fast Torque Path) Stability Control (ESC) Slow Path (also EscTqWhIPtMaxF Maximum Powertrain Clutch Torque Requested by Stability Control (ESC) Fast Path (also EscTqWhIPtMaxF
PtTqCluMinDrv Powertrain: Minimum Available Torque at Clutch for Driver	EscTqCluPtMin / EngTqIncAtCluReq Minimum Powertrain Clutch Torque Requested by Electronic Stability Contr (also EscTorqueWheelPowertrainMinimumFast/S
	TrsmTqCluMinSlow Transmission: Minimum Torque at Clutch Requested by Transmission for Shift Energy Management on Slow Path TrsmTqCluMinFast Transmission: Minimum Torque at Clutch Requested by Transmission for Shift Energy Management on Fast Path

#### Figure 4: Overview of AUTOSAR torque at clutch interfaces (1)

#### Overview of AUTOSAR torque at clutch interfaces (2)



Figure 5: Overview of AUTOSAR torque at clutch interfaces (2)





Figure 6: Overview of AUTOSAR torque loss interfaces





Figure 7: Overview of AUTOSAR torque at wheel interfaces



# **4** Architecture Overview

The following figures give an overview of the domain or functional architecture. They not necessarily give a complete picture but show the most relevant interconnections and components.



**Figure 8: Overview of Functional Architecture** 



#### Figure 9: Detail – Combustion Engine Domain Architecture

15 of 61





Figure 10: Detail – Transmission System Domain Architecture



# **5** Description of Exemplary Software Components

For being able to use and understand the standardized application interfaces a typical domain architecture was used as basis for demonstrating the signal flow. The components of this example domain architecture are described in the following.

# 5.1 Powertrain Coordinator – PTC (PtCoorr)

This composition includes all functions that coordinate the operation of the Powertrain, including:

**Powertrain operation mode** – management of states of all actuators (e.g. combustion engine, clutch(es), transmission, electric motors, etc.), including engine start / stop management (conventional & hybrid Power-trains).

**Powertrain torque coordination** – Torque coordination at Powertrain (PT) level, torque prioritisation, torque distribution for realisation at at PT level, torque reserve request for the PTC, pre-coordination of driveability functions for hybrids, Powertrain driveability filters, determination of total Powertrain losses for torque calculation, wheel torque calculation (min, max, consolidated), torque at clutch calculation (min, max, consolidated), transformation of torque set point from wheel torque to torque at clutch, transformation of torque set point from torque at clutch to torque at crankshaft, control/coordination of auxiliary drivers/actuators.

**Powertrain speed coordination** – Maximum speed limitation coordination (for protection of all PT components from damage from over speed) and coordination of idle speed / engine speed set point requests from all sources, e.g. transmission.

**Powertrain ratio coordination** – all transmission ratio set point logic. Note that realisation of ratio set point is carried out by transmission system, not PTC.

# 5.2 Transmission System (Trsm)

This composition includes all functions of the transmission system, including:

**Transmission system coordination** – Determines the torque and speed ratio over transmission, converter and differential, including the calculation of torque losses in the transmission system. Coordinates mechanical protection of the Drivetrain (gearbox, driveshafts, etc.), including calculation of torque limitation.



For manual transmission, this function includes the determination of the current gear and clutch status.

**Transmission** – Management of particular states in the transmission, including shift transition, driving off situation, creeping mode etc.. In case of shift transition, this functionality calculates torque requests to optimise the transition.

Control of transmission actuators to adjust the gear to the target gear (or to adjust the gear ratio to the target gear ratio in case of CVT). Gear ratio means the theoretical / physical ratio belonging to each gear and not any actual measured value. Control of gearbox countershaft (low/higher range) actuators is not included.

Calculates the torque gain of a hydrodynamic converter and the torque required to the converter input side in idle, etc. and controls clutch or converter actuators.

All functionality related to the protection of the transmission, including calculation of torque limitation, measurement or calculation of gearbox oil temperature, etc., and calculation of requests to other systems.



#### Example of signal flow of gear signals during an single upshift or downshift

Figure 11: Example of signal flow of gear signals during a single upshift or downshift

**Drivetrain Torque Distribution (DtTqDibtn) Differential Lock** – All functionality related to the differential(s), which manage the torque distri-



R4.1 Rev 2

bution between left and right wheels, for example locking of the differential. Does not include the calculation of the distribution set point.

**Drivetrain Torque Distribution (DtTqDibtn) Transfer Case** – All functionality related to the transfer case, which manages the torque distribution between front and rear wheels. Does not include the calculation of the distribution set point.

**Drivetrain Torque Distribution (DtTqDibtn) Torque vectoring axle transmission** – All functionality related to active distribution of powertrain torque to all four wheels individually. Does not include the calculation of the distribution set point.

For additional information on Drivetrain Torque Distribution (DtTqDibtn) please also refer to [4].

# 5.3 Combustion Engine (CmbEng)

This composition includes all functions directly related to the operation and control of the vehicle's combustion engine. The following sections, 5.3.1 to 5.3.3 inclusive, define the components as a result of Combustion Engine functionality decomposition agreed to date.

# 5.3.1 Engine Speed and Position (EngSpdAndPosn)

Functions that provide all parameters linked to engine shaft position and speed, including the synchronisation on between crankshaft and cam-shaft.

Crankshaft and camshaft signal acquisition.

Calculation of the engine position.

Calculation of the relative camshaft position for systems with variable valve timing and/or lift.

Related diagnosis and plausibility checks.

5.3.2 Engine Torque Mode Management (EngTqModMngt)

Includes calculation of engine torque set point, realisation of that set point (coordination of air / fuel / ignition, etc.), determination of consolidated engine torque, control of engine speed (idle / off-idle / limitation), and management of engine modes (including overall mode, modes for realisation of engine start & stop, and combustion modes).



# 5.3.3 Combustion Engine: Miscellaneous (CmbEngMisc)

Combustion Engine Misc gathers together miscellaneous engine interfaces. In general these are common data required for correct operation of the engine (engine temperature, ambient air pressure and battery voltage) or required for fail-safe actions (crash status). The way in which these interfaces are used is not standardised In future AUTOSAR releases, it is likely that these interfaces may be moved to different (more appropriate) provider or receiver components / compositions.

# 5.4 Vehicle Motion relevant for Powertrain (VehMtnForPt)

This composition includes Powertrain functions related to vehicle motion. The following sections, 5.4.1 to 5.4.3 inclusive, define the components that have so far been agreed as part of this composition.

5.4.1 Driver Request (DrvReq)

Driver-specific conversion of accelerator pedal position to requested torque: determines the driver request related to the motion of the vehicle. For longitudinal motion, this functionality interprets the driver request as a torque request.

#### 5.4.2 Accelerator Pedal Position (AccrPedIPosn)

The component calculates a percentage from the acquired position of the sensor, and contains plausibility checks to ensure the information. Kick-down detection is included in this component.

# 5.4.3 Safety Vehicle Speed Limitation (VehSpdLimnForSfty)

Hard limitation of vehicle speed by engine torque reduction, without any comfort functionality.

5.4.4 Vehicle Motion (Powertrain): Miscellaneous (VehMtnForPtMisc)

VehMtnForPtMisc gathers together miscellaneous interfaces in the context of vehicle motion powertrain. The way in which these interfaces are used is not standardised. In future AUTOSAR releases, it is likely that these interfaces may be moved to different (more appropriate) provider



or receiver components / compositions. It is even not excluded that they are moved to components that already exist.

VehMtnForPtMisc e.g. is used to close open interfaces in the case that it is committed that some component within vehicle motion powertrain will request or provide it but it is not yet decided which component or the component is missing.

#### 5.5 Powertrain: Miscellaneous (PtMisc)

PtMisc gathers together miscellaneous powertrain interfaces. The way in which these interfaces are used is not standardised. In future AUTOSAR releases, it is likely that these interfaces may be moved to different (more appropriate) provider or receiver components / compositions. It is even not excluded that they are moved to components that already exist. PtMisc e.g. is used to close open interfaces in the case that it is committed that some component within powertrain will request or provide it but it

is not yet decided which component or the component is missing.



# 6 Sensor Actuator Design Pattern

The Sensor Actuator Design Pattern describes how to handle software (SW) device drivers to control sensors or actuators that are connected to an ECU. It is not specific for the powertrain domain but can also be applied to any other domain, e.g. chassis.

Note: In AUTOSAR release 4.1, there are mismatches in the standardized application interfaces. E.g. chassis domain Explanatory Document [4], chapter 2.5.4.1. Internal state sensors shall be synchronized in a future release.

The pattern also includes naming pattern for ports and components. The pattern is described on two levels of abstraction:

"Blackbox": Blackbox means in this context that the inner architectural splitting is not displayed. Any splitting within the Blackbox is possible, but the mandatory ports of the Blackbox have to be provided.

Note: Within the blackbox figures we used the icon for "Sensor/Actuator SW-C", however, the way of implementing it, whether as a composition or an atomic is not normalized within the blackbox pattern.

"Whitebox": Whitebox means in this context that the inner architectural splitting according to the descriptions below has to be followed.

Note: Although compositions are shown they are only to be seen as example compositions. I.e. it is not normalized whether there is one large device abstraction composition or a composition for each sensor or actuator resp. a.s.o.

Device Abstraction as defined in this document is part of the application software (ASW). The Device Abstraction is located above the RTE. It is a set of software components that abstracts from the sensors and actuators connected to a specific ECU.

The interfaces between Device Abstraction and ECU Abstraction as well as between Device Abstraction and other software components of the Application Software (ASW) are not part of the pattern. They are only included to show the embedding in the overall architecture. "A sensoractuator software-component is written for a concrete sensor or actuator and uses the ECU abstraction interface [9]." The interfaces between sensor-actuator components and ECU abstraction components are client-server interfaces.



Note: The pattern does not yet cover diagnostics. To support diagnostics it might be usefull to split an atomic component into one composition containing several atomic components.

This pattern was derived based on the AUTOSAR method described in Figure 12.



Figure 12: Interfaces between Hardware and Software [6]

The name pattern uses the following syntax defined according to ANTLR [8]. This syntax and some parts used (like 'keyword') are also used in [7], e.g. in [TPS\_STDT\_00055].

Semantic port prototype (blueprint) definition together with short name pattern and long name pattern:

<u>({Prefix})ElecRaw</u> : Electrical Raw Value of {Prefix} - Unfiltered electrical raw sensor value(s). Electrical signals can only be represented in voltage, current and time [10].

({Prefix})ElecBascFild : Electrical Basic Value of {Prefix} - Basic filtered electrical raw sensor value(s) (e.g. maximum allowed phase shift is one scheduling raster or maximum 360° crankshaft rotation if exhaust gas pulsation dependent). Electrical representation of a technical signal [10]. Electrical signals can only be represented in voltage, current and time.

<u>({Prefix})Raw</u> : Raw Value of {Prefix} - physical raw/base sensor value(s). Simple conversion of basic filtered electrical (ElecBascFild) to physical value(s).

23 of 61



<u>({Prefix})Measd</u> : {Prefix} (Measured) - Final filtered and offset corrected physical sensor value(s). Physical sensor value(s)/standard sensor value(s). The physical sensor value is the linerarized/filtered physical raw/base sensor value including offset. At this step a (significant) phaseshift could be possible.

({Prefix}|Consold) : {Prefix} - Consolidated physical value, either a measured value (Measd) or a modelled value (Estimd). Final filtered and offset corrected consolidated actuator value/physical sensor value(s). Virtual physical sensor value/fused sensor value that comes as close as possible to the technical signal. In case of inability to provide a physical sensor value (e.g. failure, implausibility or other reasons) a substitute value/default value or a frozen value is provided.

<u>({Prefix})Estimd</u> : {Prefix} (Estimated) - Final filtered and offset corrected physical sensor value(s) replacement model value(s) for physical sensor value(s)/standard sensor value(s).

<u>({Prefix})Outp</u> : Output of {Prefix} - Final controller output (closed loop or open loop). E.g. torque or engine speed. Typically expressed as percentage.

({Prefix})Sp : Setpoint {Prefix} - Final actuator setpoint. Typically expressed as percentage.

<u>({Prefix})Read</u> : Requested Setpoint {Prefix} - Final required physical setpoint. Typically expressed as percentage but could also be expressed e.g. as factor.

With

Prefix : ('keyword')+ used for signal naming A typical prefix could look like this: {Comp}('Stg'{'index'})('AtBnk'{'index'}) with ('Stg'{'index'})('AtBnk'{'index'}) meaning "{'index'} stage at bank {'index'}", the two {'index'}-parts have a complete different semantic meaning but are numbers: is used if there are different sensors in usage and several sensors



of the same kind are used in different stages and banks Example: TrboChrgrStg3rdAtBnk2nd

Suffix1 : ('Stg'{'index'})('AtBnk'{'index'})

used only as part of the component prototype names (instance of

component types)

('Stg'{'index'})('AtBnk'{'index'})

means "{'index'} stage at bank {'index'}", the

two {'index'}-parts have a complete different semantic meaning but are numbers:

is used if there are different sensors in usage and several sensors of the same kind are used in different stages and banks Example: TrboChrgrStg3rdAtBnk2nd

Suffix2 : ('keyword')+

used only as part of the component type names to distinguish between

different HW components of the same type

- Comp : ({**Senso**r} | {**Actuator**})
- Sensor : ('keyword')+

Actuator : ('keyword')+

Comp/Sensor/Actuator -

name of the Actuator/Sensor (HW component)

or

name of a virtual actuator/sensor, i.e. there is not necessarily a HW component available with this name but the device abstraction provides a corresponding signal as if there would be

or

a more generic description of a set of actuators/sensors with common properties

or

HW implementation specific identifier Examples: TrboChrgr / EngT a.s.o.

'index' in this context is one of the following keywords:(1st | 2nd | 3rd | 4th | 5th | 6th | 7th | 8th | 9th )

The semantics of 'Stg' and 'Bnk' is defined as follows: 'Stg' - Index for devices connected in a row.



'AtBnk' - Index for parallel-connected devices

Semantics of 'keyword' is defined in [7], [TPS\_STDT\_00004].

'At', 'Stg' and 'Bnk' correspond to standardized keywords. Please note that not all keyword abbreviations used in names of examples are (yet) part of the standard.

Components together with name pattern

SwComponentPrototype Name and SwComponentType Name:

```
(Drvr{Device}Elec | DevDrvr{Device} | Dev{Device}Virt |
DevCoorrVirt)('keyword')*
With
Device: (Snsr | Actr)
```

Typical derived names with long name snippets (these names are also used in the figures):

SwComponentPrototypeName :

DrvrSnsrElec(For{Sensor}({Suffix1}))	// Electrical sensor
driver	
DevDrvrSnsr(For{Sensor}({Suffix1}))	// Sensor device
driver	
DevSnsrVirt(For{Sensor}({Suffix1}))	// Virtual device
driver	
DrvrActrElec(For{Actuator}({Suffix1}))	// Electrical actuator
driver	
DevDrvrActr(For{Actuator}({Suffix1}))	// Actuator device
driver	
<pre>DevCoorrVirt(For{Actuator}({Suffix1}))  </pre>	// Virtual device
coordinator	
SwComponentTypeName :	
DrvrSnsrElec(For{Sensor}({Suffix2}))	// Electrical sensor
driver	
DevDrvrSnsr(For{Sensor}({Suffix2}))	// Sensor device
driver	
DevSnsrVirt(For{Sensor}({Suffix2}))	// Virtual device
driver	
<pre>DrvrActrElec(For{Actuator}({Suffix2}))  </pre>	// Electrical actuator
driver	



driver

DevCoorrVirt(For{Actuator}({Suffix2})) coordinator

// Virtual device

Note: // are remarks

Please be aware: The {Sensor} part of the component type and the component prototype name are not necessarily the same: within the type a more generic sensor/actuator description might be choosen whereas the prototype is very precise and denotes a specific sensor/actuator. Exam-DevDrvrActrForTrboChrgrStg3rdAtBnk2nd : ple: (Prototype : Type) DevDrvrActrForStdPidCtrlr.

For compositions instantiating the atomic component types typical names are

{Comp}({Suffix1}) : {Comp}({Suffix2})

Used colours in figures:

Black mandatory

Red recommended -

Green optional -

Figure 16 presents the complete overall design pattern as it is used for a closed loop controlled actuator with position feedback.

Examples for applying the pattern are shown in Figure 17 (throttle valve) and Figure 18 (turbo charger). For turbo charger an extended example showing multiple instantiations within a 2 bank system with 3 stages is presented in Figure 19. Please note: In most cases it is not recommended to use company names in model names (like "HellaXYZ" used in the figures). Company names etc. are only used in the examples to show the difference between type and prototype and what is the reason for the difference. For general rules and recommendations how to deal with variants in models, as for example expressed by the company names in the examples, please refer to the modeling guides and templates.

Different tailored patterns derived from the overall pattern in Figure 16 for sub use cases are given:



Figure 21 presents the tailoring for a standard sensor; typical standard sensors are temperature, pressure or position sensors.

An example is given in Figure 23 (environment temperature sensor).

In Figure 22 the signal flow is shown.

Figure 20 presents the tailoring for a smart actuator.

The sensor actuator design pattern supports software sharing on different levels:

Development partner 1 might deliver the sensors together with the basic electrical driver software (DrvrSnsrElec), development partner 2 might deliver the sensor device driver software (DevDrvrSnsr) and the third partner might develop the substitute models together with the virtual device drivers (DevSnsrVirt).

There might be different suppliers for the same sensor / actuator or there might be different sensors / actuators used within one and the same system.

In some cases there might be parts of the implementation that can be reused for different sensors / actuators. Therefore the name pattern for the component type name is more generic and does not necessarily contain the sensor / actuator name.

In other cases the sensor / actuator names are not sufficient enough to make the component type names unique so an additional identifier can be added to the component type name.

In some cases, e.g. in more easy cooperation models or in full service supplier model (black box view), it might not be required to have the full structuring according to the overall pattern and some of the SW-C might be combined in a single SW-C. The full service supplier view for sensor actuator is shown in Figure 13. The same tailored views as for the overall pattern are possible as shown in Figure 14 (simple actuator) and Figure 15 (standard sensor). They only contain mandatory interfaces.

Please note: Setpoint requests from several different SW-Cs will be handled outside the Sensor Actuator design pattern. This is demonstrated as a design pattern in Figure 24.





Figure 13: Closed loop controlled actuator and position feedback (black box view)





Figure 14: Design pattern for simple actuator (black box view)



Figure 15: Design pattern for standard sensor (black box view)





Figure 16: Closed loop controlled actuator and position feedback





Figure 17: Example: Closed loop controlled throttle and position feedback





Figure 18: Example: closed loop controlled turbo charger and position feedback



Figure 19: Example: Project system configuration for turbo charger with several stages and banks – delegation ports





Figure 20: Open loop controlled actuator or smart actuator



Figure 21: Design pattern of blueprint components for standard sensor





Figure 22: Signal flow within a sensor SW-C composition





Figure 23: Example: Standard sensor for environment temperature 39 of 61 Document ID 269: AUTOSAR\_EXP\_AIPowertrain









# 7 Additional Information

# 7.1 Differences between SW-Cs and ECUs

The SW components defined in chapter 4 are not to be confused with an ECU's functionalities.

For example, a combustion engine control ECU may contain the Combustion Engine SW-C plus other SW-Cs.

# 7.2 Functional safety

Many Powertrain signals are safety-relevant, therefore

- The AUTOSAR RTE will provide reliable communication for these signals at the low level, and
- Diagnostics and safety concepts for these signals must be applied at the higher, functional level.

AUTOSAR does not provide a Safety Concept for Powertrain systems. This must be done at the project level. This means that the specified interfaces must be checked to fulfill the safety requirements on each specific project.

# 7.3 Powertrain Application Interfaces - Decisions / Assumptions

# 7.3.1 Scope

In this document only passenger cars are considered.

# 7.3.2 PTC Composition (PtCoorr)

The PTC is not an atomic AUTOSAR SW-Component. In fact its functionalities should be separated, into several sub-components. These sub-components will communicate with each other and with AUTOSAR SW-Components outside the PTC. The interfaces between the subcomponents are not in the current scope, which is restricted to the definition of main interfaces between the non-PTC components and the PTC sub-components.



7.3.3 Definition of overboost

Overboost is a state in which the maximum torque which the combustion engine can deliver is increased for a limited period of time. Depending on the engine type, this could be realised, for example, as an increase in boost pressure on a turbocharged engine.

# 7.3.4 Coordination at the vehicle level

Coordination of vehicle energy (mechanical / electrical / thermal), vehicle operation modes, vehicle personalisation, etc., should be done at the vehicle level. This is not in the scope of the Powertrain Application Interfaces.

The composition VehMtnForPtMisc was added to [2] as an interim solution for some vehicle level issues relevant to the powertrain domain.

7.3.5 PTC Arbitration between Driver and Chassis torque requests

Figure 25 and 26, unterhalb, shows how the VLC and Stability Control torques requests could be arbitrated with the Driver Request. This is just an example to illustrate the concept behind the powertrain torque request interfaces defined in [2], it is not intended to standardise the arbitration behavior in the PTC.





Figure 25: Example of possible PTC arbitration between Driver and Chassis torque requests (request based on wheel torque)



Figure 26: Example of possible PTC arbitration between Driver and Chassis torque requests (request based on clutch torque)

7.3.6 Assumptions on modeling style and naming aspects specific for powertrain domain

AUTOSAR provides a guideline for modeling and naming of model elements ([1]).

There are architectural design patterns like the sensor actuator design pattern described in chapter 6 that also include modeling and naming aspects.

In this section only additional patterns and modeling styles followed are explained to get an overall understanding of the signals standardized for the powertrain domain.

Please note: Here standardized ports or port interfaces mean standardized port prototype blueprints or port interface blueprints [7].

Kind of Modeling in general applied within powertrain domain, especially if the system modeling guideline [1] gives some freedom.



Kind of modeling or as- sumptions	Rationale
All SenderReceiverInter- faces standardized are assumed to be measura-	In earlier versions of the standard [2] the standard did not contain information about calibration and measurement.
ble.	Since R4.0.3 all data types allow measure- ment by default (see generated .arxml [3]). So our implicit assumption that all signals are measurable is fulfilled.
All ports are assumed to be optional.	Within our example components all ports are assumed to be optional. The ports are de- rived from the port prototype blueprints with the same name. It is optional per default that port prototype blueprints are allowed to be used but not necessarily used in every pro- ject.
	In previous releases without blueprints this assumption was very important because there was no variant handling done in [2]. So within the powertrain domain it was assumed that all ports are optional. Since only ports but no components are standardized this was even more important: it means that a supplier or OEM may create a single SW-C (Software Component) and use only the standardized ports that are rele- vant for this SW-C in his sw architecture.
Port interfaces are not designed to be reused: there is a 1:1 relationship	Ports are attached to SW-C. Since SW-C are not standardized only port interfaces were really subject of usage in projects up to Re-
between port and port in- terface. The port interface has the same name as the port + an additional	lease 3.1. With Release 4.0 the standardization of ports is supported by using so-called <i>PortProto-</i> <i>typeBlueprints</i> in the meta model.
index as required by the System Modeling Guide- line [1]. Exception:	However, in practice older versions of the AUTOSAR meta model are still in use and the existing tools do not yet fully support <i>PortPrototypeBlueprints</i> .
provider then the rules of	the easy introduction of the standardized ap-



R4.1 Rev 2

Kind of Modeling in general applied within powertrain domain, espe-		
cially if the system modeling guideline [1] gives some freedom.		
Kind of modeling or as- sumptions	Rationale	
other domains are re- spected.	plication interfaces within the powertrain do- main not all features of the meta model (like e.g. connectiong of compatible interfaces with different port interface short names) therefore were yet fully exploited. A second reason was that [2] does not sup- port connecting of compatible interfaces with different port interface short names.	
If a port interface contains	There were only two alternatives:	
exactly one data proto- type its name is identical to the port interface ex- cluding the trailing index.	<ul> <li>using full name</li> <li>using name "Val" for Value</li> <li>Disadvantage of solution 2) would have been that many ports would have been assumed to be compatible by tools because identical data prototype names (with compatible inter- faces) allow an automatic connection.</li> <li>Within specification tools like e.g. ASCET-SD or MATLAB/Simulink it might be possible to only show the data prototype name and to not display the port interface or port name.</li> <li>Within the powertrain domain it was an im- portant design goal to use as few data types</li> </ul>	
Computation methods were not subject to reuse yet.	The [2] does not support the reuse of compu- tation methods, only of data types. But reuse of computation methods should be consid-	
In most cases only one data prototype was de- fined per port interface.	This kind of modeling allows the biggest flex- ibility in implementing the standard.	
	On assembly level e.g. it is not allowed that the r-port has more data prototypes than the p-ports ([6], figure 6.3). Therefore in such cases, when several data prototypes are part of one r-port the data prototypes themselves cannot be assumed to be optional, only the complete r-port.	



R4 1	Rev	2
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Kind of Modeling in general applied within powertrain domain, espe-	
cially if the system modeling guideline [1] gives some freedom.	
Kind of modeling or as-	Rationale
sumptions	
	In older versions of the AUTOSAR standard is was not allowed that a sub-component provides only part of the port, i.e. if several data prototypes were part of a port interface it was implicitly standardized that there is one SW-C providing it. When splitting the infor- mation on several port interfaces each data prototype might be provided by a different SW-C.
In most cases no records were used to define port interfaces.	The rationale is similar to the one stated for the assumption to only define one <i>DataPro-</i> <i>totype</i> per <i>PortInterface</i> . Since within the powertrain domain <i>PortInter-</i> <i>faces</i> with multiple <i>DataPrototypes</i> are sel- domly used, there is no necessity to use rec- ords and it is assumed that timing related aspects of the data prototypes are to be handled separately, that there are other more flexible possibilities to do so.

Specific Naming Assumptions for Powertrain Domain		
Assumption	Rationale	
The names were chosen	In Powertrain ECUs there are thousands of	
such that automatic gen-	calibration relevant data. So it is important	
eration of display names	to apply the System Modeling Guideline [1]	
is possible.	in a way that automatic generation of dis-	
	play names is possible.	
	See [5] for details.	
In general the keyword	Reason for doing so is to have short names	
abbreviation "Consold" for	whenever possible (see [TR_MCM_70020]	
"Consolidated" as well as	in [5])	
the abbreviation "Act" for	Example:	
"Actual" was suppressed.	<i>PtTqClu</i> for "Torque at Clutch "	
In case a port is only for	Reason for doing so is to have short names	
Fast Path or Slow Path	whenever possible (see [TR_MCM_70020]	
"Fast" and "Slow" is add-	in [5])	
ed. In the other cases	Example:	
"SlowFast" is suppressed.	PtTqWhIMinWoCutOff affects Slow and	



Specific Naming Assumptions	s for Powertrain Domain
Assumption	Rationale
	Fast path <i>EscTqWhIPtMaxSlow</i> or <i>EscTqWhIPtMaxFast</i> is for a specific path
	For more details on fast and slow paths see chapter 3.3
"State" is used as abbre- viation "St" for "State" and "Status".	In most cases it is very difficult to explain the difference between State and Status. So for sake of simplicity and consistency within the powertrain domain only "St" is used.
TqWhl means sum of Torque Wheels whereas TqWhlInd means the torque of an individual wheel.	Within Powertrain the sum of torque wheels is more often used. So information ("Ind") is added in the case that individual wheel tor- ques are meant. Example:
	<i>PtTqWhI</i> for "Total Wheel Torque Provided by Powertrain" <i>PtTqWhIInd</i> for "Torque Delivered by the Powertrain to the Individual Wheels"
Prepositions are only used in the short name if it really helps understand- ing.	In most cases the preposition does not really add information and makes short names unnecessarily long (see [TR_MCM_70020] in [5]).
	Examples: <i>PtTqWhIMinWoCutOff</i> for Minimum Possi- ble Total Powertrain Torque at Wheels Without Complete Fuel Cut Off using the preposition 'Wo' (without)
	<i>PtTqWhlReqDrvVlc</i> for Powertrain: Total Requested Propulsion Wheel Torque by the Driver or Vehicle Longitudinal Control (VLC) instead of <i>PtTqAtWhlReqByDrvForVlc</i> as recom- mended by the system modeling guideline [1].



# 8 Appendix: Mapping Ports to Display Names - Powertrain Domain

In the following display names for the standardized port prototype blueprints are defined. It is recommended to use the display name without name space identifier "AR\_" in case no naming conflicts are expected within the system or for the ECU. In all other cases it is recommended to use the display name with name space identifier.

The rules of [5] for generating display names are followed.

There are only the following exceptions in which a name was choosen manually:

EscVWhlInd: Instead of numbers for the index element a meaningful name part was added.

EscSts: We always use "st" according to original Keywords Phys/Log (P/L-List) for Powertrain Domain.

The virtual name space for ports is the top-level package AUTOSAR abbreviated with AR\_ (see [TR\_MCM\_70040] in [5]). Virtual name spaces are described in [5].

Port prototype blueprint names are unique within top-level package "AUTOSAR".

The sub packages of the top-level package AUTOSAR partly define virtual name spaces (e.g. for data types and port interfaces) but none of these name spaces are relevant for the generation of display names.

Basis for the definition of the display names is [2], sheets "0502\*" for Powertrain.

Note: Long Names might differ in [2] (in content but not in meaning) since not all long names were yet made consistent with new sensor/actuator pattern. Consolidation with Chassis application interfaces might also lead to additional long name changes.

DisplayName	w/o	Shortname of Port / addi-	Longname of Dis-
Name	Space.	tional information if need-	playname (=
With Name	Space	ed	PortPrototypeBlue-
add AR_	before		print name extended
name,	e.g.		in case of multiple
AR_Abs_flgA	ctv		data prototypes or
			arrays)
Abs_flgActv		AbsFlgActv	Antilock Braking
			System (ABS) Con-



Explanation of Application Interfaces of the Powertrain Domain

R4.1 Rev 2

		trol Active
AccrPedl_rat	AccrPedIRat	Accelerator Pedal
		Ratio
AccrPedl_ratFild	AccrPedIRatFild	Filtered Accelerator
		Pedal Ratio
AccrPedl_ratGrdt	AccrPedIRatGrdt	Accelerator Pedal
		Ratio Gradient
Alt_tq	Altiq	Alternator Mechani- cal Torque (Load)
Alt_tqReq	AltTqReq	Requested Mechan-
		ical Torque for Alter-
		nator (Generator) at
· · - · - ·		Engine Crank Shaft
AxleFrntCoorr_st	AxleFrntCoorrSt	Status of the Front
		Axle Coordinator
AxieReCoorr_st	AxieReCoorrSt	Status of the Rear
Dott u	Dett	Axie Coordinator
Ball_U BrkDodL flaDod	Ballu	Battery Voltage
DIKPEUI_IIgPSu	DIRPEULIYSU	Pressed
BrkPedl_rat	BrkPedIRat	Brake Pedal Position
CluPedl_rat	CluPedIRat	Clutch Pedal Ratio
Drv_flgGearShiftDw	DrvFlgGearShiftDwnReq	Gear Shift Down
nReq		Request by Driver
Drv_flgGearShiftUp	DrvFlgGearShiftUpReq	Gear Shift Up Re-
Req		quest by Driver
Drv_figKdDetd	DrvFigKdDetd	Driver: Kickdown
DrvReq_ratvirtAccr	DivReqRatvinAcciPedi	Driver Request: Vir-
Peul		del Petio
DryRea taCluEast	DryPogTgCluEast	Driver Request of
Divivey_iquid asi	Diviced i doitin ast	Clutch Torque (East
		Torque Path)
DryRea taCluSlow	DryRegTaCluSlow	Driver Request of
Driftoq_iqoidoion	Brittediacien	Clutch Torque (Slow
		Torque Path)
DrvReg toWhlFast	DrvRegTgWhlFast	Driver Request of
- 1		Wheel Torque (Fast
		Torque Path)
DrvReq_tqWhISlow	DrvReqTqWhISlow	Driver Request of
		Wheel Torque (Slow



		Torque Path)
Dtd_ratTqDistbnReq WhIFrntLe	DtdRatTqDistbnReq Front Left Wheel	<ul> <li>/ Requested</li> <li>Drivetrain Torque</li> <li>Distribution - Front</li> <li>Left Wheel</li> </ul>
Dtd_ratTqDistbnReq WhIFrntRi	DtdRatTqDistbnReq Front Right Wheel	<ul> <li>/ Requested</li> <li>Drivetrain Torque</li> <li>Distribution - Front</li> <li>Right Wheel</li> </ul>
Dtd_ratTqDistbnReq WhIReLe	DtdRatTqDistbnReq Rear Left Wheel	<ul> <li>/ Requested</li> <li>Drivetrain Torque</li> <li>Distribution - Rear</li> <li>Left Wheel</li> </ul>
Dtd_ratTqDistbnReq WhIReRi	DtdRatTqDistbnReq Rear Right Wheel	<ul> <li>/ Requested</li> <li>Drivetrain Torque</li> <li>Distribution - Rear</li> <li>Right Wheel</li> </ul>
Dtd_tqDftlAxleFrntR eq	DtdTqDftlAxleFrntReq	Drivetrain Torque Distribution: Re- quested Differential Torque at Front Axle Actuator
Dtd_tqDftlAxleReRe q	DtdTqDftlAxleReReq	Drivetrain Torque Distribution: Re- quested Differential Torque at Rear Axle Actuator
Dtd_tqDftlTrfReq	DtdTqDftlTrfReq	Drivetrain Torque Distribution: Re- quested Differential Torque at Transfer Case
EgyMngt_st	EgyMngtSt	State of Energy Management
Eng_n	EngN	Engine Speed
Eng_nGearTar	EngNGearTar	Engine Speed at Target Gear
Eng_nGrdt	EngNGrdt	Engine Speed Gra- dient
Eng_nMax	EngNMax	Maximum Allowed Engine Speed
Eng_nMin	EngNMin	Minimum Allowed

- AUTOSAR confidential -



V3.1.0

R4.1 Rev 2

		Engine Speed
Eng_pAmbAir	EngPAmbAir	Engine Ambient Air
		Pressure
Eng_t	EngT	Engine Temperature
Eng_tqCrksft	EngTqCrksft	Engine Torque at Crankshaft
Eng_tqCrksftMax	EngTqCrksftMax	Maximum Engine Torque at Crank- shaft
Eng_tqCrksftMaxFa st	EngTqCrksftMaxFast	Maximum Engine Torque at Crank- shaft Fast Path
Eng_tqCrksftMaxOp tmCdn	EngTqCrksftMaxOptmCdn	Maximum Engine Torque at Crank- shaft at Optimum Conditions
Eng_tqCrksftMaxPro tn	EngTqCrksftMaxProtn	Maximum Allowed Torque at Crank- shaft for Engine Pro- tection
Eng_tqCrksftMin	EngTqCrksftMin	Minimum Engine Torque at Crank- shaft considering all engine losses
Eng_tqCrksftMinBas c	EngTqCrksftMinBasc	Minimum Engine Torque at Crank- shaft for Powertrain realized by Slow Path
Eng_tqCrksftMinFas t	EngTqCrksftMinFast	Minimum Engine Torque at Crank- shaft Fast Path
Eng_tqCrksftMinWo CutOff	EngTqCrksftMinWoCutOff	Minimum Engine Torque at Crank- shaft for Powertrain realized by Slow and Fast Path
Eng_tqDynJ	EngTqDynJ	Engine Dynamic Moment of Inertia
Eng_tqResvPt	EngTqResvPt	Torque Reserve Re- quest from Engine to Powertrain



		N4.1 NEV Z
Esc_flgNoFuCutOff	EscFlgNoFuCutOff	Request "No Fuel Cut Off" by Electron- ic Stability Control (ESC)
Esc_st	EscSts	Electronic Stability Control (ESC) Status
Esc_stShiftPrevnSta by	EscStShiftPrevnStaby	Electronic Stability Control (ESC): State of Shift Prevention for Stability Reasons
Esc_tqCluPtMaxFas t	EscTqCluPtMaxFast	Maximum Power- train Clutch Torque Requested by Elec- tronic Stability Con- trol (ESC) Fast Path
Esc_tqCluPtMaxSlo w	EscTqCluPtMaxSlow	Maximum Power- train Clutch Torque Requested by Elec- tronic Stability Con- trol (ESC) Slow Path
Esc_tqCluPtMin	EscTqCluPtMin	Minimum Powertrain Clutch Torque Re- quested by Electron- ic Stability Control (ESC)
Esc_tqWhIPtMaxFa st	EscTqWhIPtMaxFast	Maximum Power- train Wheel Torque Requested by Elec- tronic Stability Con- trol (ESC) Fast Path
Esc_tqWhIPtMaxSlo w	EscTqWhIPtMaxSlow	Maximum Power- train Wheel Torque Requested by Elec- tronic Stability Con- trol (ESC) Slow Path
Esc_tqWhIPtMinFas t	EscTqWhIPtMinFast	Minimum Powertrain Wheel Torque Re- quested by Electron- ic Stability Control (ESC) Fast Path
Esc_tqWhIPtMinSlo w	EscTqWhIPtMinSlow	Minimum Powertrain Wheel Torque Re-



R4.1 Rev 2

		R4.1 REV Z
		quested by Electron- ic Stability Control (ESC) Slow Path
Esc_vMax	EscVMax	Maximum Vehicle Speed due to Elec- tronic Stability Con- trol (ESC)
Esc_vVhlIndFrntLe	EscVWhlInd / Index 0	Electronic Stability Control (ESC): Vec- tor of Individual Speed of Wheels - Front Left
Esc_vVhlIndFrntRi	EscVWhlInd / Index 1	Electronic Stability Control (ESC): Vec- tor of Individual Speed of Wheels - Front Right
Esc_vVhlIndReLe	EscVWhlInd / Index 2	Electronic Stability Control (ESC): Vec- tor of Individual Speed of Wheels - Rear Left
Esc_vVhlIndReRi	EscVWhlInd / Index 3	Electronic Stability Control (ESC): Vec- tor of Individual Speed of Wheels - Rear Right
Hvac_tq	HvacTq	Mechanical Torque for A/C Compressor
Hvac_tqReq	HvacTqReq	Requested Mechan- ical Torque for A/C Compressor
Outd_t	OutdT	Outdoor Tempera- ture
Pt_flgAltDeactvt	PtFlgAltDeactvt	Powertrain: Request to Deactivate Alter- nator (Generator)
Pt_flgDrvOvrdVlc	PtFlgDrvOvrdVlc	Powertrain: Driver Request Overrides Vehicle Longitudinal Control (VLC)
Pt_flgEngRun	PtFlgEngRun	Powertrain: Engine



R4.1 Rev 2

		is Running
Pt_flgEngStop	PtFlgEngStop	Powertrain: Engine
Pt flaEnaStonRea	PtFlaEnaStonRea	Powertrain: Engine
		Stop Request
Pt_flgEngStopReqAl lwd	PtFlgEngStopReqAllwd	Powertrain: Request to Stop Engine is Allowed
Pt_flgEngStrtReq	PtFlgEngStrtReq	Powertrain: Engine Start Request
Pt_flgEngStrtReqAll wd	PtFlgEngStrtReqAllwd	Powertrain: Request to Start Engine is Allowed
Pt_flgHvacDeactvt	PtFlgHvacDeactvt	Powertrain: Request to Deactivate Air Conditioner (A/C)
Pt_flgNoTqWhlReq	PtFlgNoTqWhlReq	Powertrain: No Torque Request for Wheel Torque
Pt_flgTqDecPsbl	PtFlgTqDecPsbl	Powertrain: Torque Decrease Possible
Pt_flgTqIncPsbl	PtFlgTqIncPsbl	Powertrain: Torque Increase Possible
Pt_nClu	PtNClu	Powertrain: Speed at Clutch
Pt_nEngSp	PtNEngSp	Powertrain: Engine Speed Setpoint
Pt_ratTqDistbnReqF rntLe	PtRatTqDistbnReq	Powertrain: Re- quested Percental Distribution of Torque to Wheels - Front Left Wheel
Pt_ratTqDistbnReqF rntRi	PtRatTqDistbnReq	Powertrain: Re- quested Percental Distribution of Torque to Wheels - Front Right Wheel
Pt_ratTqDistbnReqR eLe	PtRatTqDistbnReq	Powertrain: Re- quested Percental Distribution of Torque to Wheels - Rear Left Wheel



R4.1	Rev	2

Pt_ratTqDistbnReqR eRi	PtRatTqDistbnReq	Powertrain: Re- quested Percental Distribution of Torque to Wheels - Rear Right Wheel
Pt_tqClu	PtTqClu	Powertrain: Torque at Clutch
Pt_tqCluDyn	PtTqCluDyn	Powertrain: Dynamic Torque at Clutch
Pt_tqCluMaxDrv	PtTqCluMaxDrv	Powertrain: Maxi- mum Available Torque at Clutch for Driver
Pt_tqCluMinDrv	PtTqCluMinDrv	Powertrain: Mini- mum Available Torque at Clutch for Driver
Pt_tqCluReqWoTrs mIntv	PtTqCluReqWoTrsmIntv	Powertrain: Torque at Clutch Request Without Transmis- sion Intervention
Pt_tqCluTarGear	PtTqCluTarGear	Powertrain: Predict- ed Torque at Clutch at Target Gear
Pt_tqCluWoTrsmIntv	PtTqCluWoTrsmIntv	Powertrain: Torque at Clutch Without Transmission Inter- vention
Pt_tqCrksftReqFast	PtTqCrksftReqFast	Crankshaft Torque Request to be real- ized by the Fast Path of Powertrain
Pt_tqCrksftReqSlow	PtTqCrksftReqSlow	Crankshaft Torque Request to be real- ized by the Slow Path of Powertrain
Pt_tqMaxAlt	PtTqMaxAlt	Powertrain: Maxi- mum Allowed Me- chanical Load for Alternator (Genera- tor) at Engine Crank Shaft.



Explanation of Application Interfaces of the Powertrain Domain

V3.1.0

R4.1	Rev 2	2
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Pt_tqMaxHvac	PtTqMaxHvac	Powertrain: Maxi- mum Allowed
		Torque for A/C
		Compressor
Pt_tqMaxSteer	PtTqMaxSteer	Powertrain: Maxi-
		mum Allowed Me-
		chanical Load for
		Steering at Engine
		Crank Shaft
Pt_tqMaxTrsmOilPm	PtTqMaxTrsmOilPmp	Maximum Allowed
р		I ransmission Oil
		Pump Load
Pt_tqResvEng	PtiqResvEng	Torque Reserve Re-
		train to Engine
Pt_ta\Whl	PtTaWhl	Total Wheel Torque
		Provided by Power-
		train
Pt_tqWhlIndFrntLe	PtTqWhlInd	Torque Delivered by
		the Powertrain to the
		Individual Wheels -
		Front Left Wheel
Pt_tqWhlIndFrntRi	PtTqWhlInd	Torque Delivered by
		the Powertrain to the
		Individual vyneels -
Pt_ta\//blindRel.e	PtTaWbllnd	Torque Delivered by
		the Powertrain to the
		Individual Wheels -
		Rear Left Wheel
Pt_tqWhlIndReRi	PtTqWhlInd	Torque Delivered by
		the Powertrain to the
		Individual Wheels -
		Rear Right Wheel
Pt_tqWhlMax	PtTqWhIMax	Powertrain: Maxi-
		mum Possible Total
		I orque at wheels
	FUQVIIIVIAXDIV	Total Torquo of
		Wheels at Ontimum
		Conditions Delivered
		by Powertrain for



R4.1 Rev 2

		Driver
Pt_tqWhlMin	PtTqWhIMin	Minimum Available Total Powertrain Torque at Wheels
Pt_tqWhIMinDrv	PtTqWhIMinDrv	Minimum Possible Total Torque at Wheels Delivered by Powertrain for Driver
Pt_tqWhlMinWoCut Off	PtTqWhIMinWoCutOff	Minimum Possible Total Powertrain Torque at Wheels Without Complete Fuel Cut Off
Pt_tqWhIReq	PtTqWhIReq	Powertrain: Total Requested Propul- sion Wheel Torque
Pt_tqWhIReqDrvVlc	PtTqWhIReqDrvVIc	Powertrain: Total Requested Propul- sion Wheel Torque by the Driver or Ve- hicle Longitudinal Control (VLC)
Ssm_flgGearParkRe q	SsmFlgGearParkReq	Standstill Manager: Request to Engage the Parking Lock
Steer_tq	SteerTq	Hydraulic Power Steering Load at Engine Crank Shaft
Steer_tqReq	SteerTqReq	Requested Hydraulic Power Steering Load at Engine Crank Shaft
TrfCaseCoorr_st	TrfCaseCoorrSt	Status of the Trans- fer Case Coordinator
TrsmClu_stAct	TrsmCluStAct	Transmission: Actual Clutch State
TrsmClu_stTar	TrsmCluStTar	Transmission: Tar- get State of the Clutch
Trsm_flgCtrsftActv	TrsmFlgCtrsftActv	Transmission: Coun- tershaft Active
Trsm_flgDtOpen	TrsmFlgDtOpen	Transmission:

- AUTOSAR confidential -



R4.1 Rev 2

		Drivetrain Opened
Trsm_flgParkLockE ngd	TrsmFlgParkLockEngd	Transmission: Park Lock Engaged
Trsm_flgShiftProgs	TrsmFlgShiftProgs	Transmission: Flag indicates that a Gear Shift is In Progress
Trsm_flgSptMod	TrsmFlgSptMod	Transmission: Sport Mode Request by Driver
Trsm_flgWntrMod	TrsmFlgWntrMod	Transmission: Win- ter Mode Request by Driver
Trsm_nInp	TrsmNInp	Transmission: Speed at Input
Trsm_nrGearAct	TrsmNrGearAct	Transmission: Actual Gear
Trsm_nrGearReq	TrsmNrGearReq	Transmission: Re- quested Gear
Trsm_nrGearTar	TrsmNrGearTar	Transmission: Tar- get Gear
Trsm_nrTyp	TrsmNrTyp	Transmission Type
Trsm_ratGear	TrsmRatGear	Transmission: Get the Gear Ratio of the Gear of Interest (C/S)
Trsm_ratGearAct	TrsmRatGearAct	Transmission: The Actual Gear Ratio being Currently En- gaged in the Gear Box
Trsm_ratGearReq	TrsmRatGearReq	Transmission: Re- quested Gear Ratio
Trsm_ratGearTar	TrsmRatGearTar	Transmission: The Gear Ratio which will be Engaged in the Gear Box when Tar- get Gear is Reached
Trsm_ratTqPtAct	TrsmRatTqPtAct	Transmission: Actual Powertrain Torque Ratio
Trsm_stAxelFrntActr	TrsmStAxleFrntActr	Transmission: Status of the Front Axle Ac-



R4.1 Rev 2

		tuator
Trsm_stAxelReActr	TrsmStAxleReActr	Transmission: Status of the Rear Axle Ac- tuator
Trsm_stGearLvr	TrsmStGearLvr	Transmission: Actual Gear Lever Position
Trms_stTrfCaseDftI	TrsmStTrfCaseDftl	Transmission: Status of the Transfer Case Differential
Trsm_tOil	TrsmTOil	Transmission: Oil Temperature
Trsm_tqCluMaxFast	TrsmTqCluMaxFast	Transmission: Max- imum Torque at Clutch Requested by Transmission for Shift Energy Man- agement on Fast Path
Trsm_tqCluMaxProt n	TrsmTqCluMaxProtn	Transmission: Max- imum Torque at Clutch Requested by Transmission for Gearbox Protection
Trsm_tqCluMaxSlow	TrsmTqCluMaxSlow	Transmission: Max- imum Torque at Clutch Requested by Transmission for Shift Energy Man- agement on Slow Path
Trsm_tqCluMinFast	TrsmTqCluMinFast	Transmission: Mini- mum Torque at Clutch Requested by Transmission for Shift Energy Man- agement on Fast Path
Trsm_tqCluMinSlow	TrsmTqCluMinSlow	Transmission: Mini- mum Torque at Clutch Requested by Transmission for Shift Energy Man-



R4.1 Rev 2

		agement on Slow Path
Trsm_tqDftlAxleFrnt Act	TrsmTqDftlAxleFrntAct	Transmission: Actual Differential Torque at Front Axle
Trsm_tqDftlAxleFrnt Max	TrsmTqDftlAxleFrntMax	Transmission: Max- imum Differential Torque at Front Axle
Trsm_tqDftlAxleReA ct	TrsmTqDftlAxleReAct	Transmission: Actual Differential Torque at Rear Axle
Trsm_tqDftlAxleReM ax	TrsmTqDftlAxleReMax	Transmission: Max- imum Differential Torque at Rear Axle
Trsm_tqDftlTrfAct	TrsmTqDftlTrfAct	Transmission: Actual Differential Transfer Torque
Trsm_tqDftlTrfMax	TrsmTqDftlTrfMax	Transmission: Max- imum Differential Transfer Torque
Trsm_tqOilPmp	TrsmTqOilPmp	Transmission Oil Pump Load
Trsm_tqOilPmpReq	TrsmTqOilPmpReq	Requested Trans- mission Oil Pump Load
Trsm_tqWhlIndDistb nFrntLe	TrsmTqWhlIndDistbn	Individual Torque at Wheel Distribution as realized by Transmission Sys- tem - Front Left Wheel
Trsm_tqWhlIndDistb nFrntRi	TrsmTqWhlIndDistbn	Individual Torque at Wheel Distribution as realized by Transmission Sys- tem - Front Right Wheel
Trsm_tqWhlIndDistb nReLe	TrsmTqWhlIndDistbn	Individual Torque at Wheel Distribution as realized by Transmission Sys- tem - Rear Left



R4.1 Rev 2

		Wheel
Trsm_tqWhlIndDistb nReRi	TrsmTqWhlIndDistbn	Individual Torque at Wheel Distribution as realized by Transmission Sys- tem - Rear Right Wheel
Veh_stOper	VehStOper	Vehicle Operating State
Veh_tqWhIMaxSftyL imn	VehTqWhIMaxSftyLimn	Safety Torque Limit calculated from the Vehicle's Maximum Speed
Veh_vLgt	VehVLgt	Vehicle Speed (Lon- gitudinal)
Vlc_stShiftPrevnCmf t	VlcStShiftPrevnCmft	Vehicle Longitudinal Control (VLC): State of Shift Prevention for Comfort Reasons
Vlc_tqWhIPtMax	VlcTqWhIPtMax	Maximum Torque at Wheel by Vehicle Longitudinal Control (VLC)
Vlc_tqWhIPtMin	VlcTqWhIPtMin	Minimum Torque at Wheel by Vehicle Longitudinal Control (VLC)